

Description:	Definition of reference solar combisystem, Germany
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Introduction

This document describes the reference solar combisystem for domestic hot water preparation and space heating in Germany. The system is modelled with TRNSYS to calculate the fuel consumption and electric energy needed to provide the required domestic hot water and space heating as well as the substituted fuel provided by the combisystem. Using this result the levelized costs of heating (LCoH) for the substituted fuel is calculated using Eq. 1 and the reference costs for the investment of the system, installation costs, fuel and electricity costs.

Hydraulic Scheme of the System

Key data	
Collector area	15 m ²
Heat store volume	800 l
Location	Germany, Würzburg
Hemispherical irradiance on horizontal surface	$\Sigma G_{\text{hem,hor}} = 1118.8 \text{ kWh}/(\text{m}^2 \text{ a})$
Lifetime of system	20 years

Levelized Cost of Heat (LCoH)

LCoHs solar part without VAT	0.206 €/kWh
LCoHc conventional part without VAT	0.119 €/kWh
LCoHo complete system without VAT	0.140 €/kWh

Details of the System

Location	Germany, Würzburg
Type of system	combisystem
Weather data including - Hemispherical irradiance on horizontal surface - Beam irradiance on horizontal surface - Diffuse irradiance on horizontal surface - Ambient temperature in hourly values	test reference year (TRY Würzburg) $\Sigma G_{\text{hem,hor}} = 1118.8 \text{ kWh}/(\text{m}^2 \text{ a})$ $\Sigma G_{\text{beam,hor}} = 550.1 \text{ kWh}/(\text{m}^2 \text{ a})$ $\Sigma G_{\text{diff,hor}} = 568.7 \text{ kWh}/(\text{m}^2 \text{ a})$ $T_{\text{amb,av}} = 9.0 \text{ }^\circ\text{C}$
Collector orientation - Collector tilt angle to horizontal - South deviation of collector - Ground reflectance - Resulting hemispherical irradiance on tilted surface	45 ° south = 0° 0.2 $\Sigma G_{\text{hem,tilt}} = 1229.8 \text{ kWh}/(\text{m}^2 \text{ a})$
Load information including - Heat demand space heating - Tapping profile - Store heat losses - Tapping temperature - Average inlet temperature of cold water - Cold water inlet temperature amplitude	9090 kWh/a /1/ EU-tapping profile L (4254 kWh/a) /2/ 2041 kWh 55°C according EU tapping profile 10°C 0 K
Collector information based on gross area	TRNSYS-type 132
Number of collectors	6
Collector area of one collector	2.5 m ²
Maximum collector efficiency	0.684
Incidence angle modifier for direct irradiance b_0	0.2
Incidence angle modifier for diffuse Irradiance	0.91
Linear heat loss coefficient	3.51 W/(m ² K)
2nd order heat loss coefficient	0.011 W/(m ² K ²)
Effective heat capacity	8.0 kJ/(m ² K)
Heat store parameters	TRNSYS-type 340
Heat store volume	800 l
Auxiliary volume for DHW preparation	424 l
Store inner diameter	0.79
Rel. height of solar inlet	0.4
Rel. height of solar outlet	0.04
Rel. height of auxiliary inlet	0.9
Rel. height of auxiliary outlet	0.47
Rel. height of space heating inlet	0.2
Rel. height of space heating outlet	0.45
Rel. height of cold water inlet	0.05
Rel. height of hot water outlet	0.95

Rel. height of sensor for aux. charging	0.6
Rel. height of sensor for space heating preheating	0.45
Rel. height of sensor for collector loop	0.2
Set temperature for DHW	62.5 °C
Temperature difference space heating preheat on	4 K
Temperature difference space heating preheat off	2 K
Overall heat loss capacity rate of store	4.4 W/K
Effective vertical conductivity	1.2 W/(mK)
Heat transfer capacity rate of solar loop HX	$(kA)_{WT,Sol} = 165,9 \cdot \dot{m}^{0,283} \cdot \vartheta_m^{0,524}$ [W/K]
Heat transfer capacity rate of hot water HX	$(kA)_{WT,HW} = 75,8 \cdot \dot{m}^{0,252} \cdot \vartheta_m^{1,026}$ [W/K]
Volume solar loop HX	11.8 l
Volume hot water HX	38.5 l
Maximum heat store temperature	90 °C
Ambient temperature of heat store	15 °C
Solar thermal controller and hydraulic piping	
Total pipe length of collector loop	20 m
Inner diameter of collector loop pipe	20 mm
Temperature difference collector start-up	6 K
Temperature difference collector shut-off	4 K
Electric power of solar thermal controller	3 W
Operating hours of solar thermal controller per year	8760 h
Electric consumption of controller per year	26.3 kWh
Electric power of solar loop pump	55 W
Operating hours of solar loop pump	1073 h
Electric consumption of solar loop pump	59 kWh
Electric consumption of other el. components	-
Conventional system	
Type of auxiliary heating	Gas condensing boiler
Boiler capacity	19 kW
Mass flow	1090 kg/h (delT = 15 K)
Efficiency factor of boiler	0.9
Electric power of controller	3 W
Operating hours of controller per year	8760
Electric consumption of controller per year	26.3 kWh
Electric power of pump	55 W
Operating hours of pump (aux. heating + space heating)	3987 h
Electric consumption of pump per year	219 kWh
Investment costs I_0	5500 €

Investment costs solar thermal system	
Solar thermal collector, heat store, solar thermal controller solar thermal hydraulic components	8000 € /5/
Installation	2000 € /5/
Credit conventional heat store and share of installation	-1000 €
Overall investment costs solar thermal part I_0	9000 €
Operation costs conventional part per year	
Auxiliary heat demand hot water	3743 kWh/a
Fuel demand hot water	4159 kWh/a
Heat demand space heating	7506 kWh/a
Fuel demand space heating	8340 kWh/a
Fuel demand hot water + space heating E_t	12499 kWh/a
Cost per kWh fuel (gas)	0.066 € kWh/a /4/
Fuel costs	825 €/a
Electricity demand	246 kWh/a
Cost per kWh electric energy /4/	0.254 €
Electricity costs	62 €/a
Maintenance costs	200 €/a /3/
Gas meter	130 €/a /3/
Yearly operation and maintenance cost conventional part C_t	1217 €
Operation costs solar part per year	
Electricity demand	85 kWh/a
Cost per kWh electric energy /4/	0.254 €
Electricity costs	22 €/a
Maintenance costs ($I_0 \cdot 2\%$)	180 €/a
Yearly operation and maintenance cost solar part C_t	202 €/a
Fractional energy savings with credit for 150l-store, $UA=2,05 \text{ W/K}$	20.2 %
Saved final energy (year t) E_t	3162 kWh
Type of incentives	None
Amount of incentives	0 €
Lifetime of system	20 year
Discount rate r	0 %
Inflation rate	0 %
Corporate tax rate TR	0 %
Asset depreciation (year t) DEP_t	0 €
Subsidies and incentives (year t) S_t (considered in I_0)	0 €

Residual value RV	0 €
Discount rate r	0 %
VAT rate	19 %

Calculation of levelized cost LCoH:

$$LCoH = \frac{I_0 - S_0 + \sum_{t=1}^T \frac{C_t(1 - TR) - DEP_t \cdot TR}{(1 + r)^t} - \frac{RV}{(1 + r)^T}}{\sum_{t=1}^T \frac{E_t}{(1 + r)^t}} \quad (1)$$

Where:

$LCoH$: levelized cost of heat in €/kWh

I_0 : initial investment in €

S_0 : subsidies and incentives in €

C_t : operation and maintenance costs (year t) in €

TR : corporate tax rate in %

DEP_t : asset depreciation (year t) in €

RV : residual value in €

E_t : saved final energy (year t) in kWh

r : discount rate in %

T : period of analysis in years

Annex: Comparison to figures published in Solar Heat Worldwide

To compare the above presented LCoHs based on the saved final energy with the $LCoH_{SHWW}$ presented in Solar Heat World Wide based on the collector yield the following table is presented

Collector yield (year t) E_t	4541 kWh
$LCoH_{SHWW}$ solar part without VAT	0.124 €/kWh

References

[1] EN 12977-2 (2012): "Thermal solar systems and components – Custom built systems – Part 2: Test methods for solar water heaters and combisystems".

[2] COMMISSION DELEGATED REGULATION (EU) No 812/2013, ANNEX VII.

[3] Hafner, B. (2016): "E-Mail". Dated 13.06.2016.

[4] Check24 (2016): "Würzburg reference costs". URL: www.check24.com (accessed in Sept. 2016).

[5] Mean values of evaluated invoices, supplied by Bafa.

[6] Louvet, Y., Fischer, S. et. al. (2017): "IEA SHC Task 54 Info Sheet A1: Guideline for levelized cost of heat (LCoH) calculations for solar thermal applications". URL: <http://task54.iea-shc.org/>.

[7] Louvet, Y., Fischer, S. et.al. (2017): “Entwicklung einer Richtlinie für die Wirtschaftlichkeitsberechnung solarthermischer Anlagen: die LCoH Methode“. Symposium Thermische Solarenergie, Bad Staffelstein.

¹ To avoid confusion with the results of other works ([1], [8], [9]) also using the notion of LCoH for solar thermal systems, new acronyms were introduced in this Info Sheet. As previous studies have considered different assumptions for the definition of the terms of the LCoH equation, it does not make sense to compare the values they obtained with the LCoHs, LCoHc and LCoHo values defined here. A detailed explanation of the differences between the approaches chosen in the framework of IEA-SHC Task 54 and in the Solar Heat Worldwide report [9] can be found in Info Sheet A13 [10].